

USEFUL EQUATIONS FOR THE ABIH EXAMINATIONS

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[Metric (SI) equations are in brackets]

VENTILATION

$$Q = VA \quad V_1A_1 = V_2A_2 \quad TP = VP + SP \quad SP_1 + VP_1 = SP_2 + VP_2 + \sum \text{losses}_{1-2} \quad SP_h = -\left((F_h + 1)VP_d \right)$$

$$V = 4005 \sqrt{\frac{VP}{df}} \quad \left[V = 1.29 \sqrt{\frac{VP}{df}} \right] \quad VP = \left(\frac{V}{4005} \right)^2 df \quad \left[VP = \left(\frac{V}{1.29} \right)^2 df \right] \quad \text{hood entry loss} = F_h x VP_d$$

$$C_e = \sqrt{\frac{VP}{|SP_h|}} \quad VP_r = \left(\frac{Q_1}{Q_3} \right) VP_1 + \left(\frac{Q_2}{Q_3} \right) VP_2 \quad Q = 4005(C_e) \sqrt{\frac{|SP_h|}{df}} (A) \quad \left[Q = 1.29(C_e) \sqrt{\frac{|SP_h|}{df}} (A) \right]$$

$$Q = 4005 C_e A \sqrt{|SP_h|} \quad Q_{corr} = Q_{lower} \sqrt{\frac{SP_{gov}}{SP_{lower}}} \quad Q' = \frac{Q}{m_i} \quad t_2 - t_1 = -\frac{V_r}{Q'} \ln \left(\frac{C_{g2}}{C_{g1}} \right)$$

$$\ln \left(\frac{G - Q' C_{g2}}{G - Q' C_{g1}} \right) = -\frac{Q'(t_2 - t_1)}{V_r} \quad Q = \frac{(403)(SG)(ER)(m_i)(10^6)}{(MW)(C_g)} \quad \left[Q = \frac{(24)(SG)(ER)(m_i)(10^6)}{(MW)(C_g)} \right]$$

$$N_{changes} = \frac{60Q}{V_r} \quad C_{g2} = \frac{G \left(1 - e^{-\left(\frac{Q' \Delta t}{V_r} \right)} \right)}{Q'} \quad C_{g2} = C_{g1} e^{-\left(\frac{Q' \Delta t}{V_r} \right)} \quad Q_2 = Q_1 \left(\frac{d_2}{d_1} \right)^3 \left(\frac{RPM_2}{RPM_1} \right)$$

$$P_2 = P_1 \left(\frac{d_2}{d_1} \right)^2 \left(\frac{RPM_2}{RPM_1} \right)^2 \quad PWR_2 = PWR_1 \left(\frac{d_2}{d_1} \right)^5 \left(\frac{RPM_2}{RPM_1} \right)^3 \quad FSP = SP_{out} - SP_{in} - VP_{in} \quad FTP = TP_{out} - TP_{in}$$

NOISE

$$SPL \text{ or } L_p = 20 \log \left(\frac{P}{P_0} \right) \quad L_I = 10 \log \left(\frac{I}{I_0} \right) \quad SPL_2 = SPL_1 + 20 \log \left(\frac{d_1}{d_2} \right) \quad L_w = 10 \log \left(\frac{W}{W_0} \right)$$

$$W_0 = 10^{-12} \text{ watts} \quad L_{eq} = 10 \log \left(\frac{1}{T} \sum_{i=1}^N \left(10^{\frac{L_i}{10}} t_i \right) \right) \quad L_{PT} = 10 \log \left(\sum_{i=1}^N 10^{\frac{L_{pi}}{10}} \right) \quad TL = 10 \log \left(\frac{1}{\tau} \right)$$

$$L_p = L_w - 20 \log r - 0.5 + DI + CF \quad [L_p = L_w - 20 \log r - 11 + DI + CF] \quad DI = 10 \log Q$$

$$\%D = 100 \left(\frac{C_1}{T_1} + \frac{C_2}{T_2} + \dots + \frac{C_i}{T_i} \right) \quad T_p = \frac{T_c}{2(L_{AS} - L_c/ER)} \quad TWA_{eq} = 10 \log \left(\frac{\%D}{100} \right) + 85 \text{ dBA}$$

$$TWA = 16.61 \log \left(\frac{\%D}{100} \right) + 90 \text{ dBA} \quad f = \frac{(N)(RPM)}{60} \quad f = \frac{c}{\lambda} \quad f_2 = 2f_1 \quad f_c = \sqrt{f_1 f_2} \quad f_2 = \sqrt[3]{2} f_1$$

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GENERAL SCIENCES, STATISTICS, STANDARDS

$$ppm = \frac{V_{contam}}{V_{air}} \times 10^6 \quad ppm = \frac{P_v}{P_{atm}} \times 10^6 \quad ppm = \frac{mg/m^3 \times 24.45}{m.w.} \quad \frac{P_1 V_1}{nRT_1} = \frac{P_2 V_2}{nRT_2} \quad V_{TS} = \frac{gd_p^2 (\rho_p - \rho_a)}{18\eta}$$

$$R_e = \frac{\rho dv}{\eta} \quad \log \frac{I_o}{I} = abc \quad pH = -\log_{10}[H^+] \quad K_a = \frac{[H^+]x[A^-]}{[HA]} \quad K_b = \frac{[BH^+]x[OH^-]}{[B]}$$

$$P_{total} = X_1 P_1 + X_2 P_2 + \dots + X_i P_i \quad \text{vapor/hazard ratio} = \frac{\text{sat. concentration}}{\text{exposure guideline}} \quad TLV_{mix} = \frac{C_1}{TLV_1} + \frac{C_2}{TLV_2} + \dots + \frac{C_n}{TLV_n}$$

$$TLV_{mix} = \frac{1}{\frac{F_1}{TLV_1} + \frac{F_2}{TLV_2} + \dots + \frac{F_n}{TLV_n}} \quad RF = \frac{8}{h} x \frac{24 - h}{16} \quad RF = \frac{40}{h_w} x \frac{168 - h_w}{128} \quad C_{asb} = \frac{(C_s - C_b)A_c}{1000A_f V_s}$$

$$C_{asb} = \frac{EA_c}{1000V_s} \quad E_{fiber\ density} = \frac{\frac{f}{N_f} - \frac{B}{N_b}}{A_f} \quad d = \frac{0.61\lambda}{\eta \sin \alpha} \quad SD = \sqrt{\frac{\sum(\bar{x} - x_i)^2}{n - 1}} \quad GM = \sqrt[n]{(x_1)(x_2)\dots(x_n)}$$

$$GM = 10^{\frac{\sum(\log x)}{n}} \quad GSD = \frac{84.13\% \text{ tile value}}{50\% \text{ tile value}} \quad GSD = \frac{50\% \text{ tile value}}{15.87\% \text{ tile value}} \quad SAE = 1.645 CV_{total} \quad CV = \frac{SD}{\bar{X}}$$

$$E_c = \sqrt{E_1^2 + E_2^2 + \dots + E_n^2} \quad t = \frac{\bar{x}_1 - \bar{x}_2}{SD_{pooled} \sqrt{\frac{1}{n_1} + \frac{1}{n_2}}} \quad SD_{pooled} = \sqrt{\frac{(n_1 - 1)SD_1^2 + (n_2 - 1)SD_2^2}{n_1 + n_2 - 2}}$$

$$LCL = \frac{C_A}{PEL} - \frac{SAE \sqrt{T_1^2 C_1^2 + T_2^2 C_2^2 + \dots + T_n^2 C_n^2}}{PEL(T_1 + T_2 + \dots + T_n)} \quad RWL = LCxHMxVMxDMxAMxFMxCM \quad LI = \frac{L}{RWL}$$

$$90\% \text{ Conf Interval} = \bar{X} \pm 1.645 \frac{SD}{\sqrt{n}} \quad 95\% \text{ Conf Limit} = \bar{X} + 1.645 \frac{SD}{\sqrt{n}}$$

HEAT STRESS

$$WBGT = 0.7t_{nwb} + 0.2t_g + 0.1t_{db} \quad WBGT = 0.7t_{nwb} + 0.3t_g \quad \Delta S = (M - W) \pm C \pm R - E$$

$$Q_s = \frac{H_s}{1.08x\Delta T} \quad \left[Q_s = \frac{H_s}{20x\Delta T} \right]$$

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RADIATION

$$I_2 = I_1 \left(\frac{d_1}{d_2} \right)^2 \quad \text{Rem} = (\text{RAD})(QF) \quad D = \frac{\Gamma A}{d^2} \quad A = A_i (0.5)^{\frac{t}{T_{1/2}}} \quad A_i = \frac{0.693}{T_{1/2}} N_i \quad A = A_i e^{-\frac{0.693t}{T_{1/2}}}$$

$$I = (1/2)^A I_0 \quad I = (1/10)^B I_0 \quad I_2 = \frac{I_1}{\frac{X}{2HVL}} \quad I_2 = \frac{I_1}{\frac{X}{10TVL}} \quad X = 3.32 \log \left(\frac{I_1}{I_2} \right) (\text{HVL}) \quad I = I_0 B e^{-ux}$$

$$\frac{1}{T_{1/2\text{eff}}} = \frac{1}{T_{1/2\text{rad}}} + \frac{1}{T_{1/2\text{bio}}} \quad T_{1/2\text{eff}} = \frac{(T_{1/2\text{rad}})(T_{1/2\text{bio}})}{T_{1/2\text{rad}} + T_{1/2\text{bio}}} \quad S = \frac{E^2}{3770} \quad S = 37.7H^2 \quad S = \frac{4P}{A}$$

$$r = \left(\frac{PG}{4\pi EL} \right)^{1/2} \quad r_{\text{NHZ}} = \frac{1}{\phi} \left(\frac{4\Phi}{\pi EL} - a^2 \right)^{1/2} \quad r_{\text{NHZ}} = \frac{f_0}{b_0} \left(\frac{4\Phi}{\pi EL} \right)^{1/2} \quad r_{\text{NHZ}} = \left(\frac{\rho\Phi \cos\theta}{\pi EL} \right)^{1/2}$$

$$D_s = \frac{1}{\phi} \left(\frac{4\Phi}{\pi TL} - a^2 \right)^{1/2} \quad \text{spatial ave} = \left(\frac{\sum_{i=1}^N F S_i^2}{N} \right)^{1/2} \quad t = \frac{0.003 \text{ J/cm}^2}{E_{\text{eff}}} \quad t = \frac{EL}{ML} \times 0.1 \text{ h} \quad O.D. = \log \frac{I_0}{I}$$

$$D_L = \sqrt{a^2 + \phi^2 r^2} \quad G = 10^g / 10$$

CONSTANTS AND CONVERSIONS

$$^{\circ}\text{F} = 9/5(^{\circ}\text{C}) + 32 \quad ^{\circ}\text{R} = ^{\circ}\text{F} + 460 \quad \text{K} = ^{\circ}\text{C} + 273.15 \quad \text{molar volume at } 25^{\circ}\text{C, 1 atm} = 24.45 \text{ L} \quad 1 \text{ ft}^3 = 28.32 \text{ L}$$

$$1 \text{ ft}^3 = 7.481 \text{ U.S. gal} \quad 1 \text{ L} = 1.0566 \text{ qt} \quad 1 \text{ inch} = 2.54 \text{ cm} \quad 1 \text{ lb} = 453.6 \text{ grams} \quad 1 \text{ gram} = 15.43 \text{ grains}$$

$$1 \text{ atm} = 14.7 \text{ psi} = 760 \text{ mm Hg} = 29.92 \text{ in Hg} = 33.93 \text{ ft water} = 1013.25 \text{ mbar} = 101,325 \text{ pascals}$$

$$1 \text{ Curie} = 3.7 \times 10^{10} \text{ disint/sec (Becquerel)} = 2.2 \times 10^{12} \text{ dpm} \quad 1 \text{ Gray} = 100 \text{ Rad} \quad 1 \text{ Sievert} = 100 \text{ Rem}$$

$$1 \text{ Tesla} = 10,000 \text{ Gauss} \quad 1 \text{ BTU} = 1054.8 \text{ joules} = 0.293 \text{ watt hr} \quad 1 \text{ cal} = 4.184 \text{ joules}$$

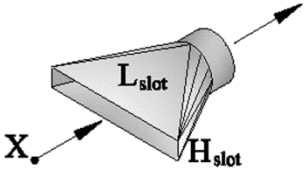
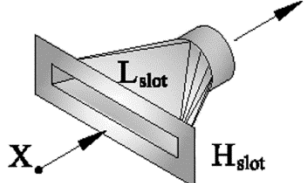
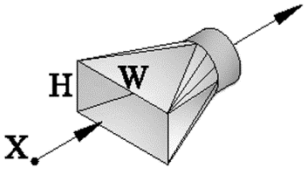
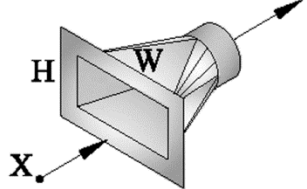
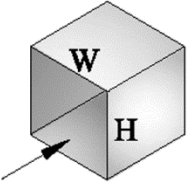
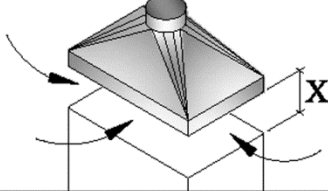
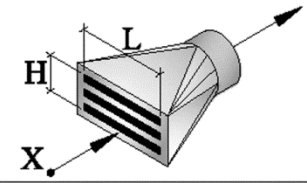
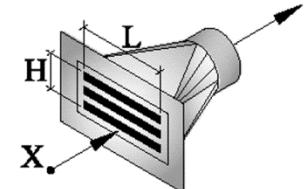
$$\text{speed of sound in air at } 68^{\circ}\text{F (} 20^{\circ}\text{C)} = 1130 \text{ fps (} 344 \text{ m/s)} \quad \text{speed of light} = 3 \times 10^8 \text{ m/s}$$

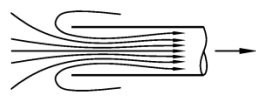
$$\text{Planck's constant} = 6.626 \times 10^{-27} \text{ erg sec} \quad \text{Avogadro's number} = 6.024 \times 10^{23}$$

$$\text{gas constant, R} = 8.314 \text{ J/mole K} = 0.082 \text{ L atm/mole K} \quad \text{density of air} = 1.29 \text{ g/L at atm, } 0^{\circ}\text{C}$$

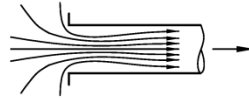
$$g = 981 \text{ cm/sec}^2 = 32 \text{ ft/sec}^2 \quad A_c = 385 \text{ mm}^2 \text{ for } 25 \text{ mm filter} \quad A_f = 0.00785 \text{ mm}^2$$

TABLE 6-3. Summary of Hood Airflow Equations

HOOD TYPE	DESCRIPTION	ASPECT RATIO, H/L	AIRFLOW
	Slot	0.2 or less	$Q = 3.7 LV_xX$
	Flanged slot	0.2 or less	$Q = 2.6 LV_xX$
	Plain opening	0.2 or greater and round	$Q = V_x(10X^2 + A_f)$ $A_f = WH$
	Flanged opening $W_f \geq \sqrt{A_f}$	0.2 or greater and round	$Q = 0.75V_x(10X^2 + A_f)$ $A_f = WH$
	Booth	To suit work	$Q = VA = V_fWH$
	Canopy	To suit work	$Q = 1.4 PVX$ P = Perimeter of work or tank X = Height above work
	Plain multiple slot opening (2) or more slots	0.2 or greater	$Q = V_x(10X^2 + A_s)$ $A_s = HL$
	Flanged multiple slot opening (2) or more slots	0.2 or greater	$Q = 0.75V_x(10X^2 + A_s)$ $A_s = HL$

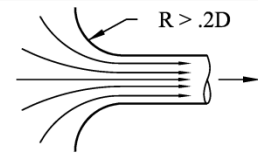


$F_h = 0.93$
 $C_e = 0.72$
PLAIN DUCT END

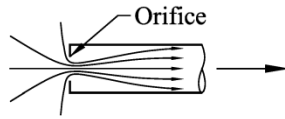


Conical
 $F_h = 0.56$
 $C_e = 0.80$
FLANGED DUCT END

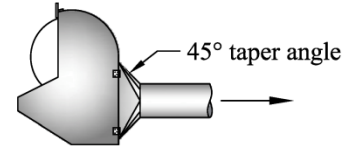
Rectangular
 $F_h = 0.64$
 $C_e = 0.78$



$F_h = 0.04$
 $C_e = 0.98$
BELLMOUTH ENTRY



$F_h = 1.78$
 $C_e = 0.60$
SHARP-EDGED ORIFICE

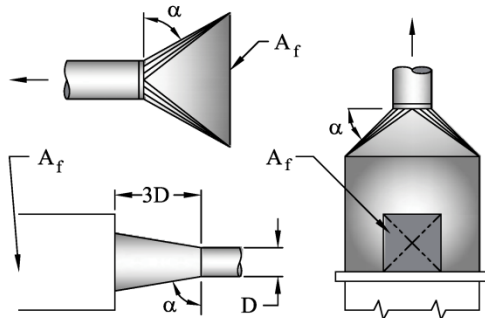


$F_h = 0.4$ (tapered take-off)
 $C_e = 0.85$
 $F_h = 0.65$ (no taper)
 $C_e = 0.78$
STANDARD GRINDER HOOD

FIGURE 9-a

TAPERED HOODS

Flanged or unflanged; conical, square or rectangular. α is the major angle on rectangular hoods.

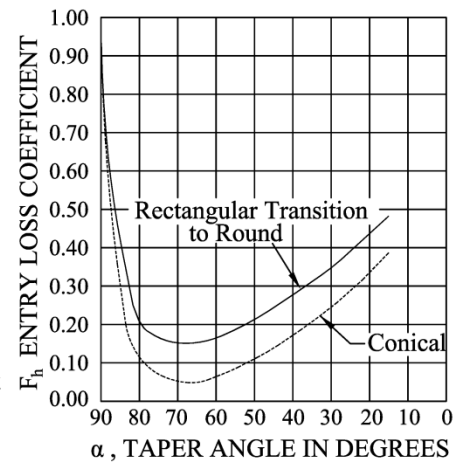


Face area (A_f) at least 2 times the duct area.

F_h Entry Loss Coefficient

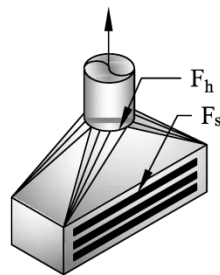
α	Conical	Rectangular
90	0.93	0.93
80	0.11	0.19
70	0.06	0.13
60	0.08	0.16
50	0.12	0.21
45	0.15	0.25
40	0.18	0.28
30	0.25	0.35
20	0.34	0.43
10	0.44	0.52
0	0.56	0.64

Note: 0° values represent round ducts butted into back of booth or hood without a rectangular to round transition.



COMPOUND HOODS

A compound hood, such as the slot/plenum shown to the right, would have 2 losses, one through the slot and the other through the transition into the duct. The slot entry loss coefficient, F_s , would have a value typically in the range of 1.00 to 1.78 (see Chapters 6 and 13). The duct entry loss coefficient is given by the above data for tapered hoods.



Hood Entry Loss =
 $F_s VP_s + F_h VP_d$
 (See Chapter 6)

MISCELLANEOUS VALUES

HOOD	ENTRY LOSS COEFFICIENT F_h	HOOD FLOW COEFFICIENT C_e
Abrasive blast chamber	1.00	0.50
Abrasive blast elevator	2.30	0.31
Abrasive separator	2.30	0.31
Elevators (enclosures)	1.00	0.50
Flanged pipe plus close elbow	0.80	0.56
Plain pipe plus close elbow	1.60	0.38



TITLE

HOOD ENTRY LOSS FACTORS

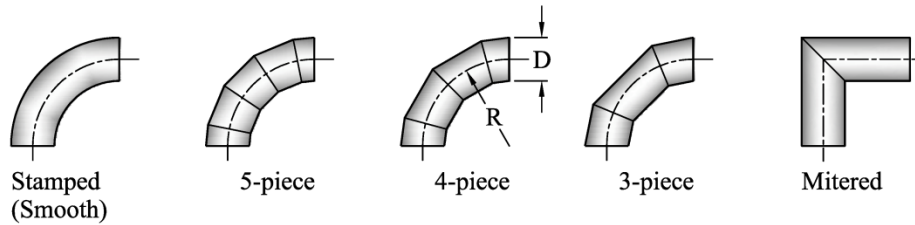
FIGURE

9-a

DATE

1-16

CHECK CODES, REGULATIONS, AND LAWS (LOCAL, STATE, AND NATIONAL) TO ENSURE THAT DESIGN IS COMPLIANT.



	R / D				
	0.75	1.00	1.50	2.00	2.50
Stamped	0.33	0.22	0.15	0.13	0.12
5-piece	0.46	0.33	0.24	0.19	0.17*
4-piece	0.50	0.37	0.27	0.24	0.23*
3-piece	0.54	0.42	0.34	0.33	0.33*

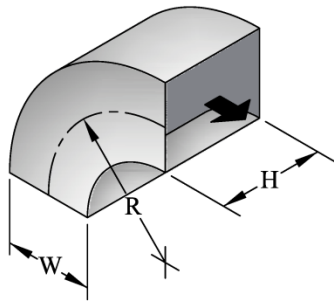
* extrapolated from published data

OTHER ELBOW LOSS FACTORS

Mitered, no vanes	1.2
Mitered, turning vanes	0.6
Flatback (R/D = 2.5)	0.05 (see Chapter 5, Figure 5-18)

NOTE: Loss factors are assumed to be for elbows of "zero length." Friction losses should be included to the intersection of centerlines.

ROUND ELBOW LOSS FACTORS



R / W	Aspect Ratio, H/W					
	0.25	0.5	1.0	2.0	3.0	4.0
0.0 (Mitered)	1.50	1.32	1.15	1.04	0.92	0.86
0.5	1.36	1.21	1.05	0.95	0.84	0.79
1.0	0.45	0.28	0.21	0.21	0.20	0.19
1.5	0.28	0.18	0.13	0.13	0.12	0.12
2.0	0.24	0.15	0.11	0.11	0.10	0.10
3.0	0.24	0.15	0.11	0.11	0.10	0.10

SQUARE & RECTANGULAR ELBOW LOSS FACTORS



TITLE

DUCT DESIGN DATA
ELBOW LOSS FACTORS

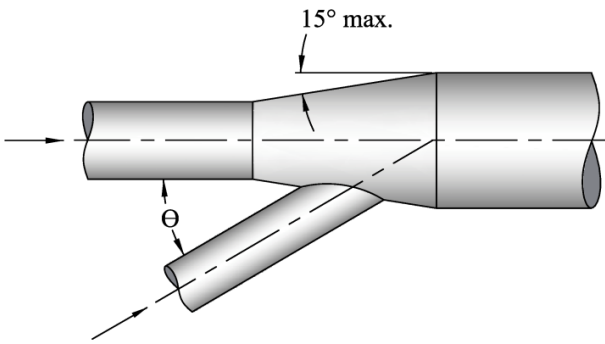
FIGURE

9-e

DATE

1-16

CHECK CODES, REGULATIONS, AND LAWS (LOCAL, STATE, AND NATIONAL)
TO ENSURE THAT DESIGN IS COMPLIANT.

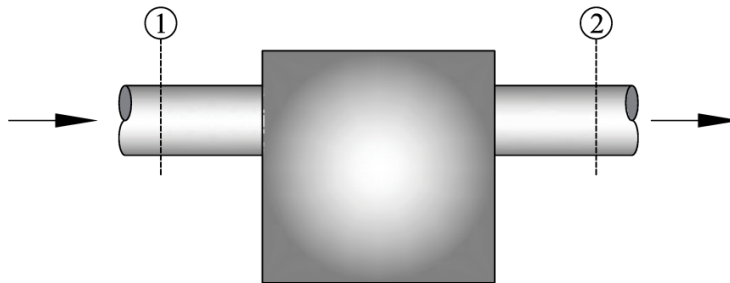


Angle θ Degrees	Loss Factor
10	0.06
15	0.09
20	0.12
25	0.15
30	0.18
35	0.21
40	0.25
45	0.28
50	0.32
60	0.44
90	1.00

Note: Branch entry loss assumed to occur in branch and is so calculated.

Do not include a regain calculation for branch entry enlargements.

BRANCH ENTRY LOSS FACTORS



$$SP_2 - SP_1 = 1.5 VP_2$$

$$F_h = 1.5$$

$$C_e = 0.8$$

TRAP OR SETTLING CHAMBER



TITLE
**BRANCH ENTRY
 LOSS FACTORS AND LOSSES
 IN SETTLING CHAMBERS**

FIGURE
9-f
 DATE
1-16

CHECK CODES, REGULATIONS, AND LAWS (LOCAL, STATE, AND NATIONAL)
 TO ENSURE THAT DESIGN IS COMPLIANT.